

# Vibrations Qualifying Exam

## Study Material

The candidate is expected to have a thorough understanding of engineering vibrations topics. These topics are listed below for clarification. Not all instructors cover exactly the same material during a course, thus it is important for the candidate to closely examine the subject areas listed below. The textbook listed below is a good source for the review and study of a majority of the listed topics. One final note, the example problems made available to the candidates are from past exams and do not cover all subject material. These problems are not to be used as the only source of study material. The topics listed below should be your guide for what you are responsible for knowing.

Suggested textbook:

*Theory of Vibrations with Applications*, W. Thompson, (Prentice-Hall)

Topic areas:

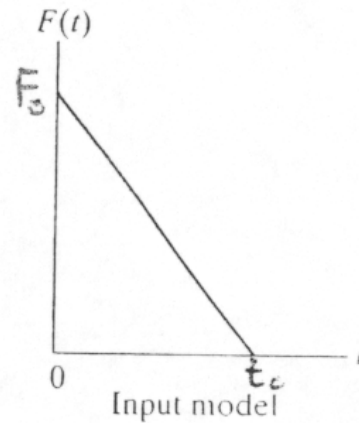
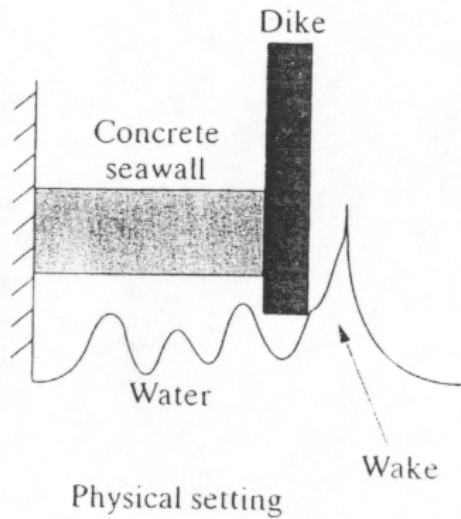
1. Oscillatory Motion
2. Free Vibration
3. Harmonically Excited Vibration
4. Transient Vibration
5. Systems with Multiple Degrees of Freedom
6. Properties of Vibrating Systems
7. Lagrange's Equations
8. Vibration of Continuous Systems



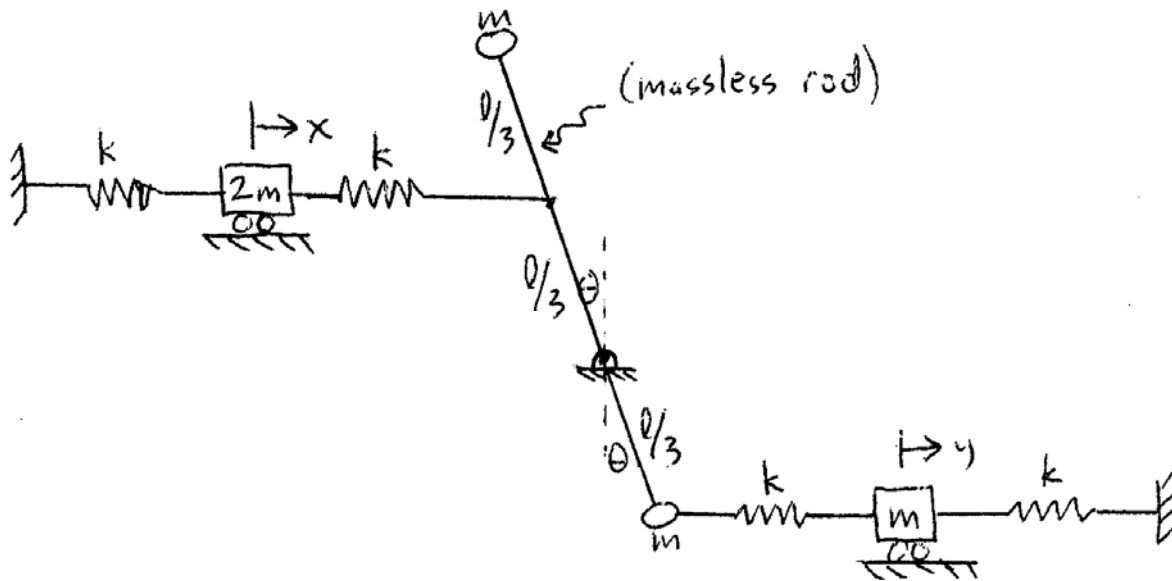
**Qualifying Exam: Vibrations**

**CLOSED BOOK**

3. A wave impacts a dike and seawall which has length  $l$ , area  $A$ , and Young's modulus  $E$ . Calculate the resulting vibration if the wave force is modeled as shown below and the seawall is modeled as an undamped single-degree-of-freedom system.



4. Use Lagrange's equations to derive the equations of motion for the following system. Obtain linearized equations about the stable equilibrium and write in matrix/vector form. Obtain the stability condition for the inverted pendulum.



NAME: \_\_\_\_\_

Spring 2006

**Qualifying Examination**  
**Subject: Vibrations**

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Work 3 of the 4 problems. Be very clear which 3 you want graded (see below). It is not acceptable to work all 4 problems and hope that the graders pick out the best worked three.

I want problems #\_\_\_\_, #\_\_\_\_, and #\_\_\_\_ graded.

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Lagrange's Equations:

$$\frac{d}{dt} \left( \frac{\partial T}{\partial \dot{q}_j} \right) - \frac{\partial T}{\partial q_j} + \frac{\partial D}{\partial \dot{q}_j} + \frac{\partial V}{\partial q_j} = Q_j \qquad \delta U = \sum Q_j \delta q_j$$

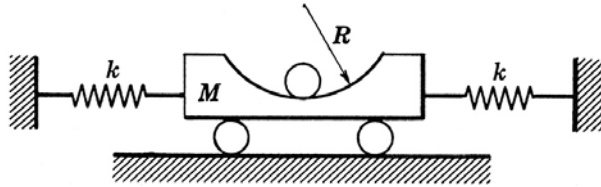
Convolution Integral:

$$x(t) = \int F(\tau)h(t-\tau)d\tau$$

## Qualifying Examination Subject: Vibrations

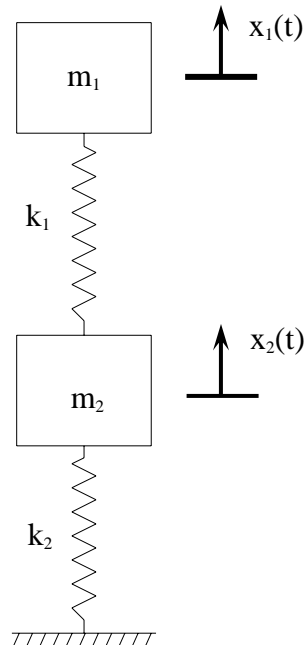
### Problem 1:

The circular cylinder has a mass  $m$  and radius  $r$ , and rolls without slipping inside the circular groove of radius  $R$ , as shown below. Derive the equations of motion.



### Problem 2:

A model of a vehicle suspension is given below. Write (a) the equations of motion in matrix form, (b) calculate the natural frequencies, and (c) determine the mode shapes for  $k_1 = 10^3$  N/m,  $k_2 = 10^4$  N/m,  $m_2 = 50$  kg, and  $m_1 = 2000$  kg.

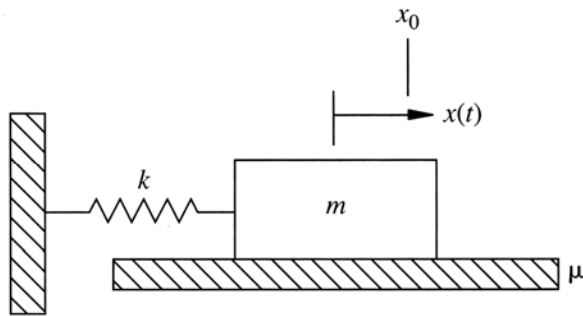


## Qualifying Examination

### Subject: Vibrations

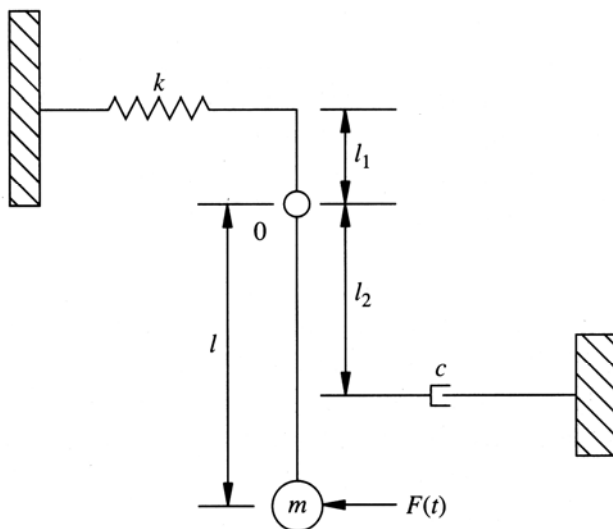
#### **Problem 3:**

The spring mass system is sliding on a surface with a kinetic coefficient of friction  $\mu$ . For the initial conditions  $x(0) = x_0$  and  $\dot{x}(0) = 0$ , determine the response of the system for one complete cycle of motion.



#### **Problem 4:**

Consider the pendulum mechanism below, which is pivoted at point O.  $k = 4 \times 10^3$  N/m,  $l_1 = 1.5$  m,  $l_2 = 0.5$  m,  $l = 1$  m, and  $m = 40$  kg. The mass of the beam is 40 kg; it is pivoted at point O and assumed to be rigid. Design the dashpot (i.e. calculate  $c$ ) so that the damping ratio of the system is 0.2. Also determine the amplitude of vibration of the steady-state response if a 10-N force is applied to the mass, as indicated in the figure, at a frequency of 10 rad/s.



NAME: \_\_\_\_\_

Fall 2004

**Qualifying Examination**  
**Subject: Vibrations**

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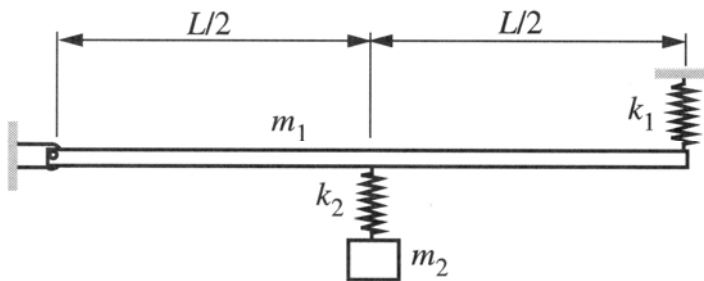
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**Problem 1:**

Determine the natural frequencies and corresponding mode shapes for the system shown below, where  $k_1 = k_2 = k$  and  $m_1 = m_2 = m$ .

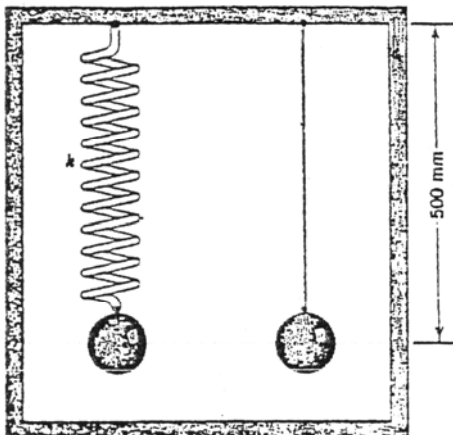


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**Problem 2:**

Two 1-kg spheres are suspended from the roof of an elevator, as shown. When the elevator is at rest the period of extensional free vibration of the spring supported sphere is the same as that for the rotation of the pendulum. Determine:

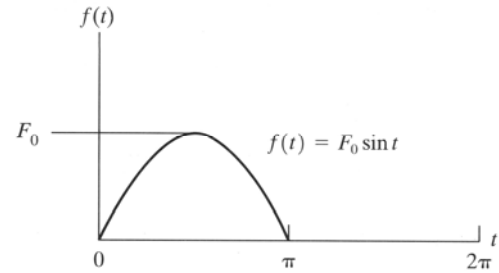
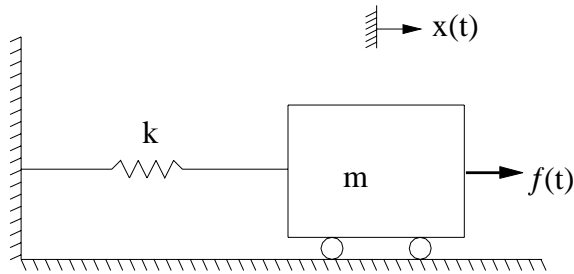
- (a) the stiffness  $k$  of the spring and
- (b) the period of each system when the elevator accelerates upward at  $a = 0.6g$ . ( $g = 9.807 \text{ m/s}^2$ )



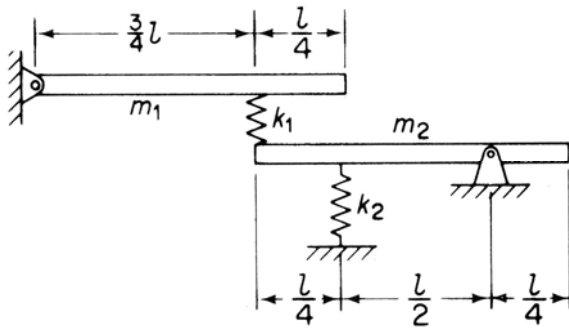
**Problem 3:**

The spring-mass mechanism shown in the figure below is subject to a force  $f(t)$  as shown. Determine the motion of the mass if the mass is at rest at the instant the force is applied.

$$x_p = \left( \frac{1}{m\omega} \right) \int_0^t F(\tau) \sin \omega(t - \tau) d\tau$$

**Problem 4:**

Using Lagrange's equations, determine the equations of motion for the two uniform bars shown. Note that they are of equal length but different masses.





NAME: \_\_\_\_\_

Fall 2003

**Qualifying Examination**  
**Subject: Vibrations**

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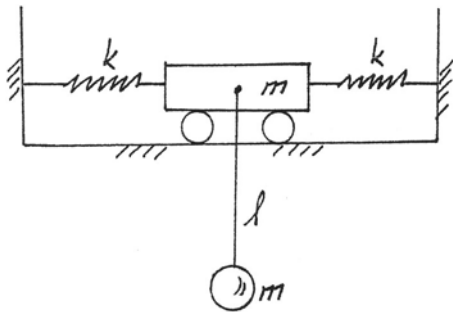
I want problems #\_\_\_\_, #\_\_\_\_, and #\_\_\_\_ graded.

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**Problem 1:**

At  $t = 0$  the block in the system is given a velocity  $v_0$  to the left. Determine the system's equations of motion and natural frequencies.

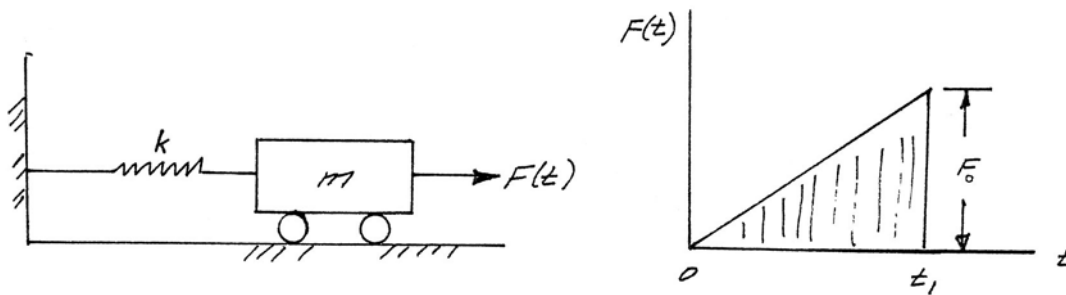


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**Problem 2:**

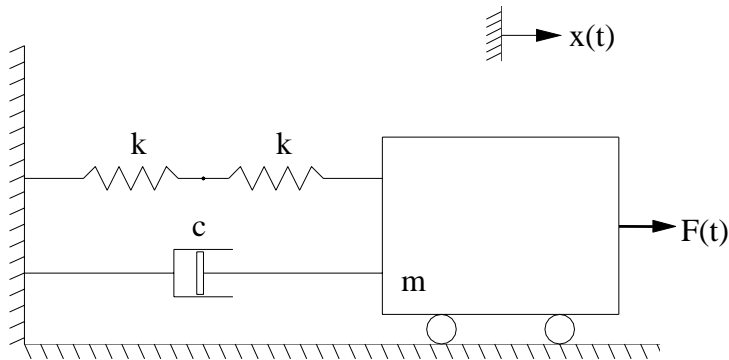
The spring-mass mechanism shown in the figure is subject to a force  $F(t)$  as shown. Determine the expression for the motion of the mass for  $t > t_1$  if the mass is at rest at the instant the force is applied.

$$x_p = \left( \frac{1}{m\omega} \right) \int_0^t F(\tau) \sin \omega(t - \tau) d\tau$$



**Problem 3:**

Consider the system below,  $m = 2\text{kg}$ ,  $k = 100\text{ N/m}$ , and  $c = 40\text{ N}\cdot\text{s/m}$ ,  $F_0 = 4\text{N}$ . Calculate the response,  $\mathbf{x(t)}$  of the system to the applied load.

Initial Conditions

$$x(0) = 0.1\text{ m}$$

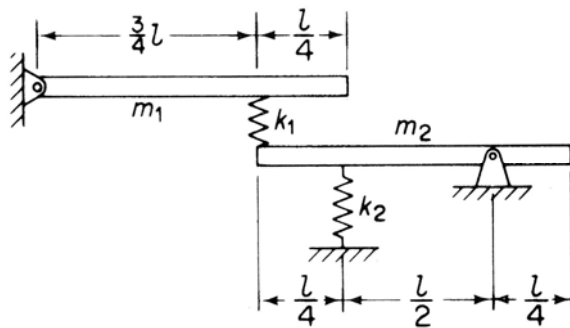
$$\dot{x}(0) = 0\text{ m/s}$$

Forcing Function

$$F(t) = 10\cos(5t) + F_0$$

**Problem 4:**

The two uniform bars shown below are of equal length but of different masses. Determine the equations of motion, natural frequencies, and mode shapes. Mass  $m_2 = 2m_1$ .



# Qualifying Exam Spring 2003

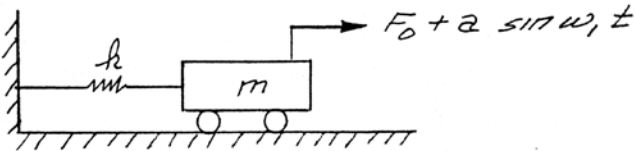
## Vibrations

This portion of the qualifying exam is **closed** book. You may have a calculator.

There are 4 problems here. Pick any 3 to work.

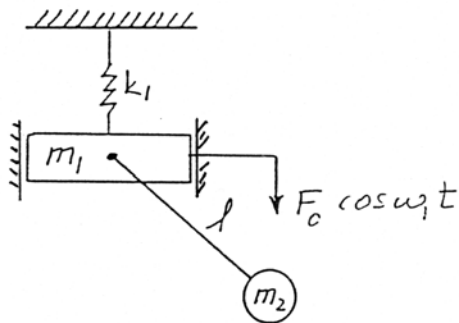
### Problem 1:

Solve for the resulting motion if the force  $F_0 + a \sin(\omega_1 t)$  is suddenly applied to the system at time  $t = 0$ . (Assume  $\omega_1^2 \neq \frac{k}{m}$ )



### Problem 2:

Find the steady state solution for the forced system shown.

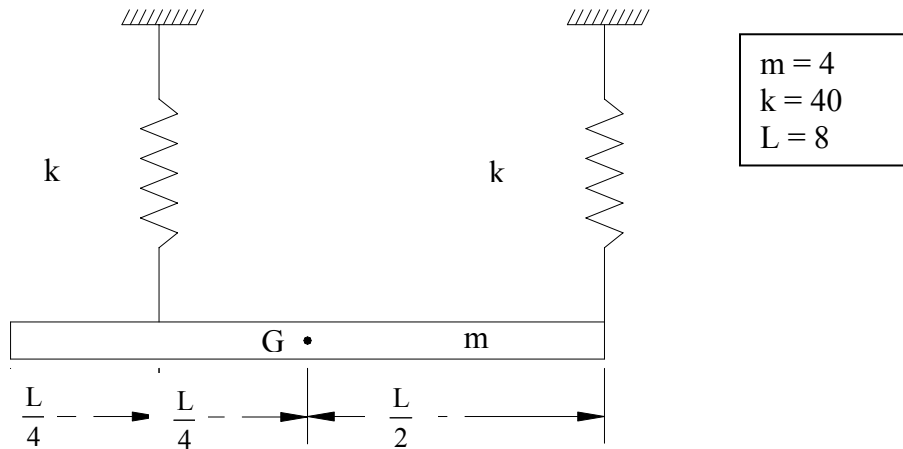


### Problem 3:

For the system below determine (a) the equations of motion in matrix form, (b) the natural frequencies, (c) the mode shapes, and (d) write the equations that describe the displacement use constants for the unknown quantities with the appropriate subscripts.

$$I = \frac{1}{12} mL^2$$

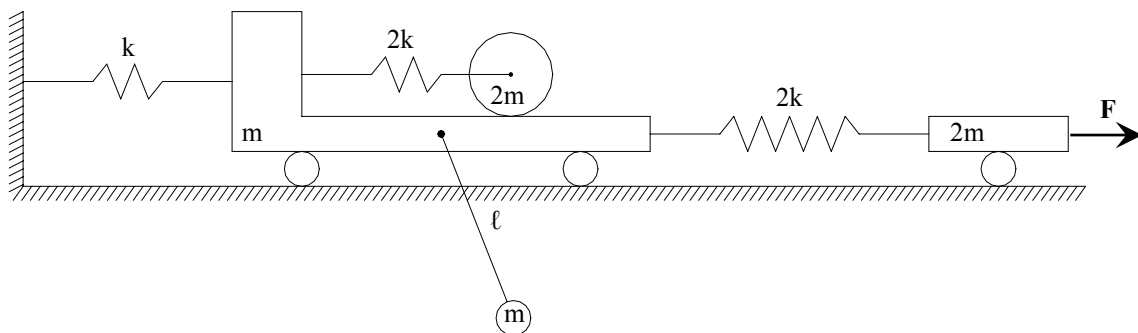
Assume that the system below is shown in its equilibrium position. Use the vertical displacement of G as one of the degrees-of-freedom.



### Problem 4:

For the system below, determine the equation of motion in matrix form using Lagrange's equation.

$$I = \frac{1}{2} mr^2 \text{ for the disk.}$$



# Qualifying Exam Fall 2002

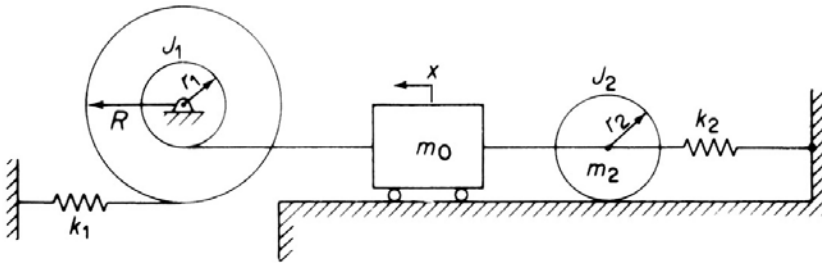
## Vibrations

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There are 4 problems here. Pick any 3 to work.

### Problem 1:

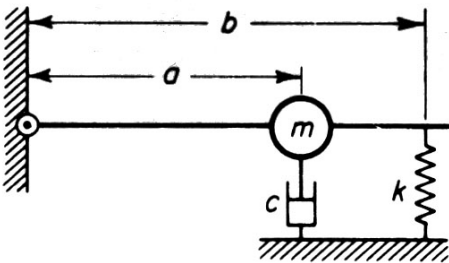
Determine the equation of motion of the system shown below in terms of the displacement  $x$ . Determine the expression for the system's natural frequency.



### Problem 2:

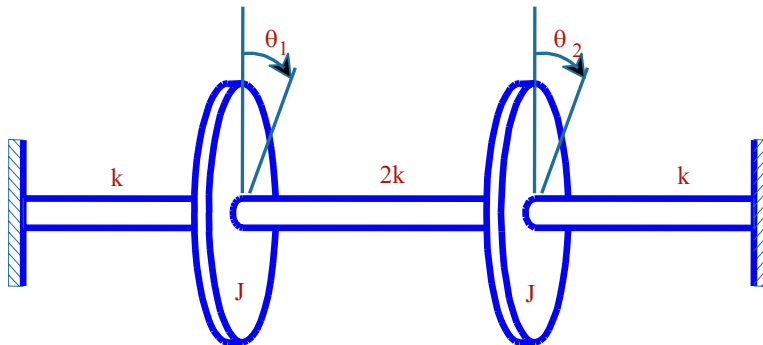
For the system shown determine:

- the equation of motion
- the natural frequency of damped oscillation
- the critical damping coefficient.



### Problem 3:

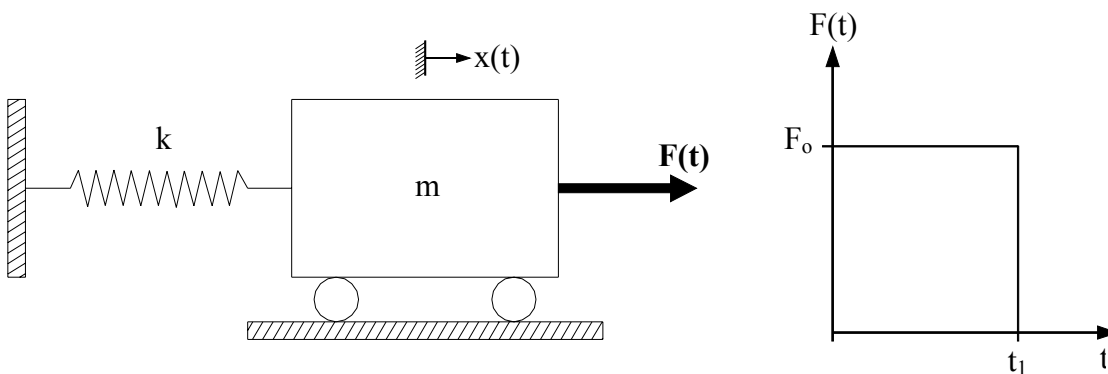
For the systems shown below, determine the natural frequencies and mode shapes. You may leave the solution in terms of  $J$  and  $k$ .



### Problem 4:

The system shown below is excited by the square pulse shown below. Determine the response of the system  $x(t)$  to the excitation pulse. The response of the system due to a unit impulse excitation at  $t = 0$  is given by:

$$h(t) = \frac{1}{m\omega_n} \sin \omega_n t$$



# Qualifying Exam Fall 1998

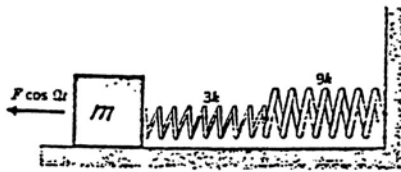
## Vibrations

This portion of the qualifying exam is **closed** book. Note: During this time the exam was a vibrations and controls exam, so the controls problems are not included.

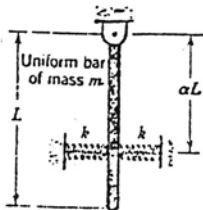
There are 4 problems here. Pick any 3 to work.

**Problem #1.** The harmonic force  $F \cos \Omega t$  is applied to the block at  $t=0$ . Initially, the block was at rest in the position where the compressive force in the series connected spring is  $F$ . Determine

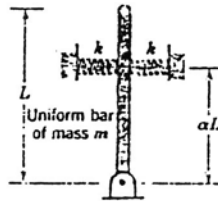
- (a) the response of the block, and
- (b) the amplitude and phase angle of the steady state response for  $\Omega = (k/m)^{1/2}$



**Problem #2.** Determine the natural frequency of each of the systems shown. Are there any values of the spring stiffness  $k$  for which either system will not oscillate freely about the vertical position?



(a)



(b)

$$I_G(\text{rod}) = mL^2/12$$

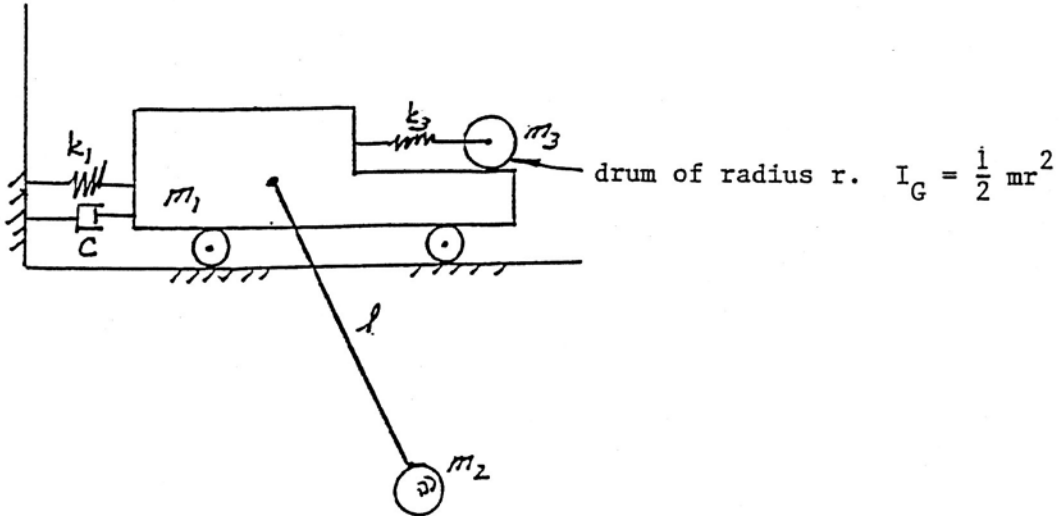
$$I = I_G + md^2$$

**Problem #3.**

Part (a). Determine the equations of motion for the system shown.

Part (b). Give a detailed outline of the steps to be performed to determine the natural frequencies and corresponding amplitudes of the system. No calculations need be shown. Neglect friction. Rod is massless. ( $m_2$  is of negligible

$$\left( \frac{d}{dt} \right) (\partial T / \partial \dot{q}_j) - \partial T / \partial q_j + \partial D / \partial \dot{q}_j + \partial V / \partial q_j = Q_j \quad \text{radius.}$$





# Qualifying Exam Spring 1997

## Vibrations

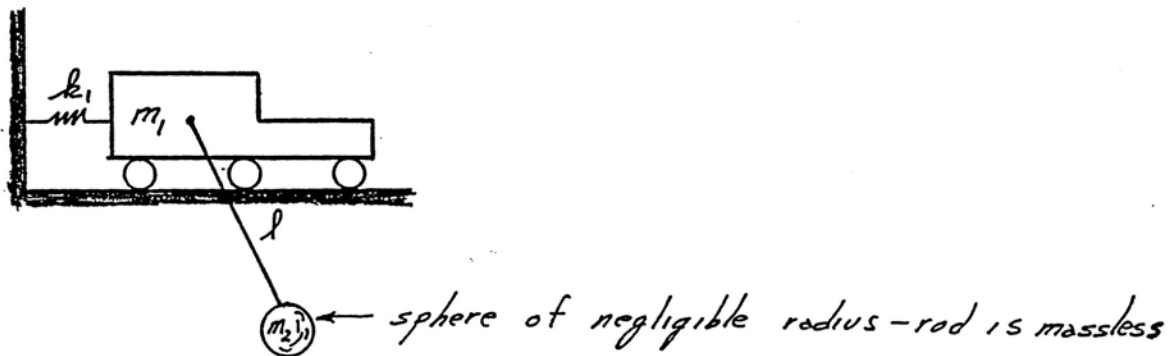
This portion of the qualifying exam is **closed** book. Note: During this time the exam was a vibrations and controls exam, so the controls problems are not included.

There are 4 problems here. Pick any 3 to work.

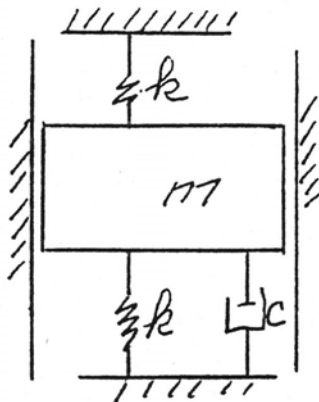
### Problem #1

For the system shown determine

- (a) the equations of motion, and
- (b) the natural frequencies and relative amplitudes.



Problem #2 Determine the equation of motion for the system shown. Find an expression for the critical damping coefficient.



# Qualifying Exam Fall 1996

## Vibrations

This portion of the qualifying exam is **closed** book. Note: During this time the exam was a vibrations and controls exam, so the controls problems are not included.

There are 4 problems here. Pick any 3 to work.

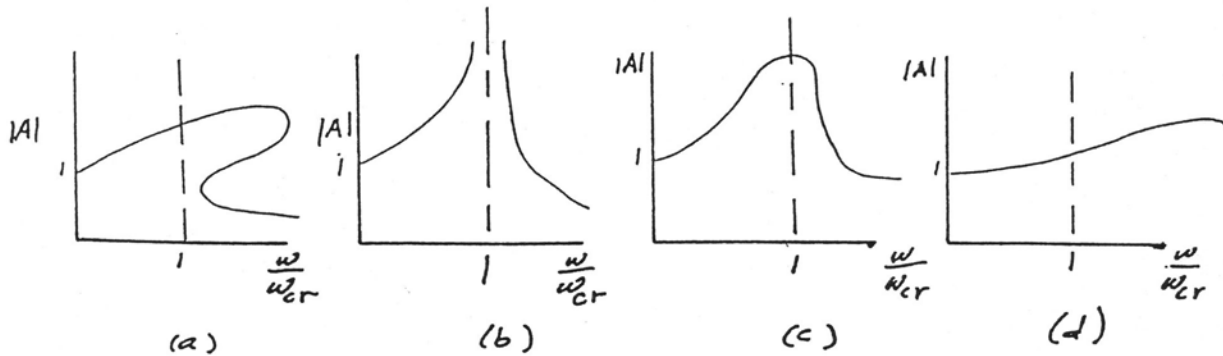
**Problem 1.** A weight attached to a spring of stiffness 3.0 lb/in has a viscous damping device. When the weight was displaced and released the period of vibration was found to be 1.80 seconds, and the ratio of consecutive amplitudes was 4.2 to 1.0. Find the following.

- (a) The logarithmic decrement.
- (b) The viscous damping factor.
- (c) the mass of the attached weight.

Note:  $\delta = \ln(x_m/x_{m+1}) = 2\pi\zeta/\sqrt{1-\zeta^2}$ ,  $\zeta = c/c_{cr}$ ,  $\omega_d = \sqrt{1-\zeta^2}\omega$

**Problem 2.** Consider parts (a) and (b) below,

- (a) The following response curves are for a one degree of freedom system. Briefly (one or two sentences) describe the most striking features of the physical system that each describes.



- (b) Briefly (one or two sentences) define:

- i. generalized coordinates
- ii. degree of freedom
- iii. beating
- iv. critical damping